WAC 51-11C-41000 Section C410-Refrigeration system requirements.

C410.1 General. Walk-in coolers, walk-in freezers, refrigerated warehouse coolers, refrigerated warehouse freezers, and refrigerated display cases shall comply with this Section.

Table C410.2

Minimum Efficiency Requirements: Commercial Refrigerators and Freezers and Refrigeration

Equipment Category	Condensing Unit Configuration	Equipment Family	Rating Temp. °F	Operating Temp. °F	Equipment Classification ^c	Maximum Daily Energy Consumption kWh/day ^{d,e}	Test Standard
Remote condensing commercial		Vertical open	38 (M)	≥32	VOP.RC.M	$0.64 \times TDA + 4.07$	AHRI 1200
		(VOP)	0 (L)	<32	VOP.RC.L	$2.20 \times TDA + 6.85$	
		Semivertical open	38 (M)	≥32	SVO.RC.M	$0.66 \times TDA + 3.18$	
		(SVO)	0 (L)	<32	SVO.RC.L	$2.20 \times TDA + 6.85$	
		Horizontal open (HZO)	38 (M)	≥32	HZO.RC.M	$0.35 \times TDA + 2.88$	
			0 (L)	<32	HZO.RC.L	$0.55 \times TDA + 6.88$	
		Vertical closed transparent (VCT)	38 (M)	≥32	VCT.RC.M	$0.15 \times TDA + 1.95$	
	Downste (DC)		0 (L)	<32	VCT.RC.L	$0.49 \times TDA + 2.61$	
refrigerators and	Remote (RC)	Horizontal closed	38 (M)	≥32	HCT.RC.M	$0.16 \times TDA + 0.13$	
freezers		transparent (HCT)	0 (L)	<32	HCT.RC.L	$0.34 \times TDA + 0.26$	
		Vertical closed	38 (M)	≥32	VCS.RC.M	$0.10 \times V + 0.26$	
		solid (VCS)	0 (L)	<32	VCS.RC.L	$0.21 \times V + 0.54$	
		Horizontal closed	38 (M)	≥32	HCS.RC.M	$0.10 \times V + 0.26$	
		solid (HCS)	0 (L)	<32	HCS.RC.L	$0.21 \times V + 0.54$	-
		Service over	38 (M)	≥32	SOC.RC.M	$0.44 \times TDA + 0.11$	
		counter (SOC)	0 (L)	<32	SOC.RC.L	$0.93 \times TDA + 0.22$	
-	Self-contained (SC)	Vertical open (VOP)	38 (M)	≥32	VOP.RC.M	$1.69 \times TDA + 4.71$	AHRI 1200
			0 (L)	<32	VOP.RC.L	4.25 × TDA + 11.82	
		Semivertical open (SVO)	38 (M)	≥32	SVO.RC.M	$1.70 \times TDA + 4.59$	
Self-contained			0 (L)	<32	SVO.RC.L	4.26 × TDA + 11.51	
refrigerators and		Horizontal open (HZO)	38 (M)	≥32	HZO.RC.M	$0.72 \times TDA + 5.55$	
commercial			0 (L)	<32	HZO.RC.L	$1.90 \times TDA + 7.08$	
without doors		Vertical closed transparent (VCT)	38 (M)	≥32	VCT.RC.M	$0.10 \times V + 0.86$	
			0 (L)	<32	VCT.RC.L	$0.29 \times V + 2.95$	
		Vertical closed solid (VCS)	38 (M)	≥32	VCS.RC.M	$0.05 \times V + 1.36$	
			0 (L)	<32	VCS.RC.L	$0.22 \times V + 1.38$	
	Self-contained (SC)	Horizontal closed transparent (HCT)	38 (M)	≥32	HCT.RC.M	$0.06 \times V + 0.37$	AHRI 1200
Self-contained			0 (L)	<32	HCT.RC.L	$0.08 \times V + 1.23$	
refrigerators and		Horizontal closed solid (HCS)	38 (M)	≥32	HCS.RC.M	$0.05 \times V + 0.91$	
commercial freezers with and without doors			0 (L)	<32	HCS.RC.L	$0.06 \times V + 1.12$	
		Service over counter (SOC)	38 (M)	≥32	SOC.RC.M	$0.52 \times TDA + 1.00$	
			0 (L)	<32	SOC.RC.L	$1.10 \times TDA + 2.10$	
Self-contained commercial refrigerators with transparent doors for pull-down temperature applications	Self-contained (SC)	Pull-down	38(M)	≥32	PD.SC.M	$0.11 \times V + 0.81$	AHRI 1200

Equipment Category	Condensing Unit Configuration	Equipment Family	Rating Temp. °F	Operating Temp. °F	Equipment Classification ^c	Maximum Daily Energy Consumption kWh/day ^{d,e}	Test Standard
Commercial ice	Remote (RC)	Vertical open (VOP)	-15 (I)	≤-5 ^b	VOP.RC.I	$2.79 \times TDA + 8.70$	AHRI 1200
		Semivertical open (SVO)			SVO.RC.I	$2.79 \times TDA + 8.70$	
		Horizontal open (HZO)			HZO.RC.I	$0.70 \times TDA + 8.74$	
		Vertical closed transparent (VCT)			VCT.RC.I	$0.58 \times TDA + 3.05$	
		Horizontal closed transparent (HCT)			HCT.RC.I	$0.40 \times TDA + 0.31$	
		Vertical closed solid (VCS)			VCS.RC.I	$0.25 \times V + 0.63$	
		Horizontal closed solid (HCS)			HCS.RC.I	$0.25 \times V + 0.63$	
		Service over counter (SOC)			SOC.RC.I	$1.09 \times TDA + 0.26$	
cream freezers	Self-contained (SC)	Vertical open (VOP)	15 (I)	≤-5 ^b	VOP.SC.I	\times TDA +	AHRI 1200
		Semivertical open (SVO)			SVO.SC.I	\times TDA +	
		Horizontal open (HZO)			HZO.SC.I	\times TDA +	
		Vertical closed transparent (VCT)			VCT.SC.I	\times TDA +	
		Horizontal closed transparent (HCT)			HCT.SC.I	\times TDA +	
		Vertical closed solid (VCS)			VCS.SC.I	\times V +	
		Horizontal closed solid (HCS)			HCS.SC.I	\times V +	
		Service over counter (SOC)			SOC.SC.I	× TDA +	

For SI: 1 square foot = 0.0929 m^2 , 1 cubic foot = 0.02832 m^3 , °C = (°F - 32)/1.8.

a The meaning of the letters in this column is indicated in the columns to the left.

b Ice cream freezer is defined in DOE 10 C.F.R. Part 431.62 as a commercial freezer that is designed to operate at or below -5°F and that the

manufacturer designs, markets or intends for the storing, displaying, or dispensing of ice cream. с

Equipment class designations consist of a combination [(in sequential order separated by periods (AAA).(BB).(C))] of:

(AAA) An equipment family code where:

- VOP = Vertical open SVO = Semi-vertical open
- HZO = Horizontal open
- VCT = Vertical transparent doors
- VCS = Vertical solid doors
- HCT = Horizontal transparent doors
- HCS = Horizontal solid doors
- SOC = Service over counter
- (BB) An operating mode code:
 - RC = Remote condensingSC = Self-contained
- (C) A rating temperature code:
 - M = Medium temperature (38°F)
 - L = Low temperature (0°F)
 - I = Ice cream temperature (15°F)
- For example, "VOP.RC.M" refers to the "vertical-open, remote-condensing, medium-temperature" equipment class.
- V is the volume of the case (ft³) as measured in AHRI 1200, Appendix C.
- ^e TDA is the total display area of the case (ft²) as measured in AHRI 1200, Appendix D.

C410.2 Commercial refrigerators, freezers and refrigerator-freezers. Refrigeration equipment, defined in DOE 10 C.F.R. Part 431.62, shall have an energy use in kWh/day not greater than the values of Table C410.2 when tested and rated in accordance with AHRI Standard 1200. The energy use shall be verified through certification under an approved certification program or, where a certification program does not exist, the energy use shall be supported by data furnished by the equipment manufacturer.

C410.2.1 Refrigerated display cases. Refrigerated display cases shall comply with the following:

1. Lighting in refrigerated display cases shall be controlled by one of the following:

1.1. Time switch controls to turn off lights during nonbusiness hours. Timed overrides for display cases shall turn the lights on for up to 1 hour and shall automatically time out to turn the lights off.

1.2. Motion sensor controls on each display case section that reduce lighting power by at least 50 percent within 3 minutes after the area within the sensor range is vacated.

2. Low-temperature display cases shall incorporate temperaturebased defrost termination control with a time-limit default. The defrost cycle shall terminate first on an upper temperature limit breach and second upon a time limit breach.

3. Antisweat heater controls shall reduce the energy use of the antisweat heater as a function of the relative humidity in the air outside the door or to the condensation on the inner glass pane.

C410.3 Walk-in coolers, walk-in freezers, refrigerated warehouse coolers and refrigerated warehouse freezers. Site-assembled and site-constructed walk-in coolers and walk-in freezers and refrigerated warehouse coolers and refrigerated warehouse freezers shall comply with the following:

1. Automatic door-closers shall be provided that fully close walk-in doors that have been closed to within 1 inch (25 mm) of full closure.

EXCEPTION: Automatic closers are not required for doors more than 45 inches (1143 mm) in width or more than 7 feet (2134 mm) in height.

2. Doorways shall be provided with strip doors, curtains, springhinged doors or other method of minimizing infiltration when doors are open.

3. Walk-in coolers and refrigerated warehouse coolers shall be provided with wall, ceiling, and door insulation of not less than R-25 or have wall, ceiling and door assembly U-factors no greater than U-0.039. Walk-in freezers and refrigerated warehouse freezers shall be provided with wall, ceiling and door insulation of not less than R-32 or have wall, ceiling and door assembly U-factors no greater than U-0.030.

EXCEPTION: Insulation is not required for glazed portions of doors or at structural members associated with the walls, ceiling or door frame.

4. The floor of walk-in coolers shall be provided with floor insulation of not less than R-25 or have a floor assembly U-factor no greater than U-0.40. The floor of walk-in freezers shall be provided with floor insulation of not less than R-28 or have a floor assembly U-factor no greater than U-0.035.

EXCEPTION: Insulation is not required in the floor of a *walk-in cooler* that is mounted directly on a slab on grade.

5. Transparent fixed window and reach-in doors for *walk-in freezer* ers and windows in *walk-in freezer* doors shall be provided with triple-pane glass, with the interstitial spaces filled with inert gas or be provided with heat-reflective treated glass.

6. Transparent fixed window and reach-in doors for *walk-in cool*ers and windows for *walk-in coolers* doors shall be provided with double-pane or triple-pane glass, with interstitial space filled with inert gas, or be provided with heat-reflective treated glass.

7. Evaporator fan motors that are less than 1 hp (0.746 kW) and less than 460 volts shall be provided with electronically commutated motors, brushless direct-current motors, or 3-phase motors.

8. Condenser fan motors that are less than 1 hp (0.746 kW) shall use electronically commutated motors, permanent split capacitor-type motors or 3-phase motors.

9. Antisweat heaters that are not provided with antisweat heater controls shall have a total door rail, glass and frame heater power draw of not greater than 7.1 W/ft^2 (76 W/m^2) of door opening for walk-in freezers and not greater than 3.0 W/ft^2 (32 W/m^2) of door opening for walk-in coolers.

10. Where antisweat heater controls are provided, they shall be capable of reducing the energy use of the antisweat heater as a function of the relative humidity in the air outside the door or to the condensation on the inner glass pane.

11. Lights in walk-in coolers, walk-in freezers, refrigerated warehouse coolers and refrigerated warehouse freezers shall either be provided with light sources with an efficacy of not less than 40 lumens per watt, including ballast losses, or shall be provided with a device that automatically turns off the lights within 15 minutes of when the walk-in cooler or walk-in freezer space is not occupied.

C410.3.1 Performance standards. Site-assembled and site-constructed *walk-in coolers* and *walk-in freezers* shall meet the requirements of Tables C410.3.1(1), C410.3.1(2), and C410.2.1(3).

Table C410.3.1(1) Walk-in Cooler and Freezer Display Doors Efficiency Requirements

Class Description	Class	Maximum Energy Consumption (kWh/day) ^a
Display door, medium temperature	DD, M	$0.04 \times A_{dd} + 0.41$
Display door, low temperature	DD, L	$0.15 \times A_{dd} + 0.29$

^a A_{dd} is the surface area of the display door.

Table C410.3.1(2) Walk-in Cooler and Freezer Nondisplay Doors Efficiency Requirements

Class Description	Class	Maximum Energy Consumption (kWh/day) ^a
Passage door, medium temperature	PD, M	$0.05 \times A_{nd} + 1.7$
Passage door, low temperature	PD, L	$0.14 \times A_{nd} + 4.8$
Freight door, medium temperature	FD, M	$0.04 \times A_{nd} + 1.9$
Freight door, low temperature	FD, L	$0.12 \times A_{nd} + 5.6$

^a A_{nd} is the surface area of the display door.

Table C410.3.1(3) Walk-in Cooler and Freezer Refrigeration Systems Efficiency Requirements

Class Description	Class	Minimum Annual Walk-in Energy Factor AWEF (Btu/hW-h)	Test Procedure
Dedicated condensing, medium temperature, indoor system	DC.M.I	5.61	AHRI 1250
Dedicated condensing, medium temperature, outdoor system	DC.M.O	7.60	
Dedicated condensing, low temperature, indoor system, net capacity $(q_{net}) < 6,500$ Btu/h	DC.L.I, < 6,500	$9.091 \times 10^{-5} \times q_{net} + 1.81$	
Dedicated condensing, low temperature, indoor system, net capacity $(q_{net}) \ge 6,500$ Btu/h	DC.L.I, ≥ 6,500	2.40	
Dedicated condensing, low temperature, outdoor system, net capacity $(q_{net}) < 6,500$ Btu/h	DC.L.O, < 6,500	$9.091 \times 10^{-5} \times q_{net} + 2.73$	
Dedicated condensing, low temperature, outdoor system, net capacity $(q_{net}) \ge 6,500$ Btu/h	DC.L.O, ≥ 6,500	3.15	
Unit cooler, medium	UC.M	9.00	
Unit cooler, low temperature, net capacity (q_{net}) < 15,500 Btu/h	UC.L, < 15,500	$9.091 \times 10^{-5} \times q_{net} + 2.73$	
Unit cooler, low temperature, net capacity $(q_{net}) \ge 15{,}500 \text{ Btu/h}$	UC.L, ≥ 15,500	4.15	

C410.4 Refrigerated case and walk-on display doors. Lighting in glass doors in all *walk-in coolers* and *walk-in freezers* and all *refrigerated warehouse coolers* and *refrigerated warehouse freezers* shall comply with the following:

1. Time switch controls to turn off lights during nonbusiness hours. Timed overrides for display cases shall turn the lights on for up to 1 hour and shall automatically time out to turn the lights off.

2. Motion sensor controls on each display case section that reduce lighting power by at least 50 percent within 3 minutes after the area within the sensor range is vacated.

C410.5 Refrigeration systems. Refrigerated display cases, *walk-in coolers* or *walk-in freezers* that are served by remote compressor and remote condensers not located in a *condensing unit*, shall comply with Sections C410.5.1, C410.5.2, and C403.9.2.3.

EXCEPTION: Systems where the working fluid in the refrigeration cycle goes through both subcritical and supercritical states (transcritical) or that use ammonia refrigerant are exempt.

C410.5.1 Condensers serving refrigeration systems. Fan-powered condensers shall comply with the following:

1. The design saturated condensing temperatures for air-cooled condensers shall not exceed the design dry-bulb temperature plus $10^{\circ}F$ (5.6°C) for low-temperature refrigeration systems, and the design dry-bulb temperature plus $15^{\circ}F$ (8°C) for medium temperature refrigeration systems where the saturated condensing temperature for blend refrigerants shall be determined using the average of liquid and vapor temperatures as converted from the condenser drain pressure.

2. Condenser fan motors that are less than 1 hp (0.75 kW) shall use electronically commutated motors, permanent split-capacitor-type motors or 3-phase motors.

3. Condenser fans for air-cooled condensers, evaporatively cooled condensers, air- or water-cooled fluid coolers or cooling towers shall reduce fan motor demand to not more than 30 percent of design wattage

at 50 percent of design air volume, and incorporate one of the following continuous variable speed fan control approaches:

3.1. Refrigeration system condenser control for air-cooled condensers shall use variable setpoint control logic to reset the condensing temperature setpoint in response to ambient dry-bulb temperature.

3.2. Refrigeration system condenser control for evaporatively cooled condensers shall use variable setpoint control logic to reset the condensing temperature setpoint in response to ambient wet-bulb temperature.

4. Multiple fan condensers shall be controlled in unison.

5. The minimum condensing temperature setpoint shall be not greater than 70° F (21°C).

C410.5.2 Compressor systems. Refrigeration compressor systems shall comply with the following:

1. Compressors and multiple-compressor system suction groups shall include control systems that use floating suction pressure control logic to reset the target suction pressure temperature based on the temperature requirements of the attached refrigeration display cases or walk-ins.

EXCEPTION:

Controls are not required for the following: 1. Single-compressor systems that do not have variable capacity capability. 2. Suction groups that have a design saturated suction temperature of 30°F (-1.1°C) or higher, suction groups that comprise the high stage of a two-stage or cascade system, or suction groups that primarily serve chillers for secondary cooling fluids.

2. Liquid subcooling shall be provided for all low-temperature compressor systems with a design cooling capacity equal to or greater than 100,000 Btu/hr (29.3 kW) with a design-saturated suction temperature of -10°F (-23°C) or lower. The subcooled liquid temperature shall be controlled at a maximum temperature setpoint of 50°F (10°C) at the exit of the subcooler using either compressor economizer (interstage) ports or a separate compressor suction group operating at a saturated suction temperature of 18°F (-7.8°C) or higher.

2.1. Insulation for liquid lines with a fluid operating temperature less than 60°F (15.6°C) shall comply with Table C403.2.10.

3. Compressors that incorporate internal or external crankcase heaters shall provide a means to cycle the heaters off during compressor operation.

C410.6 Commissioning. Refrigeration systems shall be commissioned in accordance with Section C408.

EXCEPTION: Self-contained units.

[Statutory Authority: RCW 19.27A.020, 19.27A.025, 19.27A.160 and chapters 19.27A and 19.27 RCW. WSR 22-14-091, 23-12-101, and 23-20-021, § 51-11C-41000, filed 7/1/22, 6/7/23, and 9/25/23, effective 3/15/24. Statutory Authority: RCW 19.27A.025, 19.27A.045 and chapter 19.27 RCW. WSR 20-21-080, § 51-11C-41000, filed 10/19/20, effective 2/1/21. Statutory Authority: RCW 19.27A.020, 19.27A.025, 19.27A.160 and chapter 19.27 RCW. WSR 19-24-040, § 51-11C-41000, filed 11/26/19, effective 7/1/20. Statutory Authority: RCW 19.27A.025, 19.27A.045, 19.27A.160, and 19.27.074. WSR 16-24-070, § 51-11C-41000, filed 12/6/16, effective 16-13-089, § 51-11C-41000, filed 6/15/16, 5/1/17; effective WSR Authority: RCW 19.27A.025, Statutory 19.27A.160, and 7/16/16. WSR 16-03-072, § 51-11C-41000, filed 1/19/16, effective 19.27.074. 7/1/16.]

Reviser's note: The brackets and enclosed material in the text of the above section occurred in the copy filed by the agency.